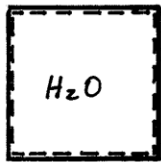


## Problem 2

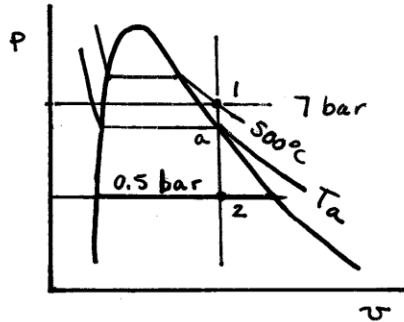
**KNOWN:** Steam is cooled in a closed rigid container from a known initial state to a known final pressure.

**FIND:** Determine the temperature at which condensation first occurs, the fraction of the total mass that condenses, and the volume occupied by saturated liquid at the final state.

**SCHEMATIC & GIVEN DATA:**



$$\begin{aligned} P_1 &= 7 \text{ bar} \\ T_1 &= 500^\circ\text{C} \\ V &= 1 \text{ m}^3 \end{aligned}$$



**ASSUMPTIONS:** (1) The steam is a closed system. (2) The volume is constant.

**ANALYSIS:** By assumptions (1) and (2), the specific volume is constant. Thus, using data from Table A-4

$$v_1 = v_a = v_2 = 0.5070 \text{ m}^3/\text{kg}$$

Then from Table A-2

$$T_a = T_{\text{sat}@ } v_a = 140^\circ\text{C}$$

Next, the fraction of the total mass that condenses is

$$\text{fraction condensed} = \frac{m_{f2}}{m} = 1 - x_2$$

$$x_2 = \frac{v_2 - v_{f2}}{v_{g2} - v_{f2}} = \frac{0.5070 - 1.0300 \times 10^{-3}}{3.420 - 1.0300 \times 10^{-3}} = 0.1480$$

Thus

$$\text{fraction condensed} = 1 - 0.1480 = 0.8520$$

$$\begin{aligned} \text{The total mass of the system is } m &= \frac{V}{v_1} = \frac{(1 \text{ m}^3)}{(0.5070 \text{ m}^3/\text{kg})} \\ &= 1.972 \text{ kg} \end{aligned}$$

Then, the volume of the liquid at state 2 is

$$V_{f2} = m v_{f2} = (1.972 \text{ kg})(1.0300 \times 10^{-3} \text{ m}^3/\text{kg})$$

$$V_{f2} = 0.00203 \text{ m}^3$$

### Problem 3

ASSUMPTIONS: 1. The closed system is shown by a dashed line on the above schematic. 2. The gas behaves as an ideal gas. 3. For the process,  $Q = W = 0$ , and kinetic and potential energy changes are negligible.

Applying the ideal gas equation of state

$$\begin{matrix} P_1 V_1 = m R T_1 \\ P_2 V_2 = m R T_2 \end{matrix} > \frac{P_2 V_2}{P_1 V_1} = \frac{T_2}{T_1} \Rightarrow P_2 = P_1 \left[ \frac{T_2}{T_1} \right] \left[ \frac{V_1}{V_2} \right]$$

The final temperature is required. Applying an energy balance

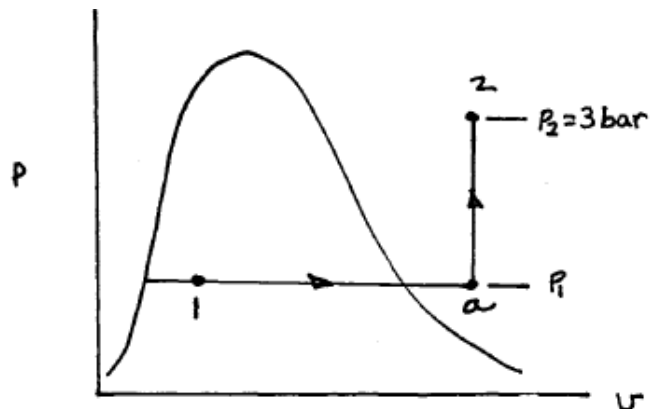
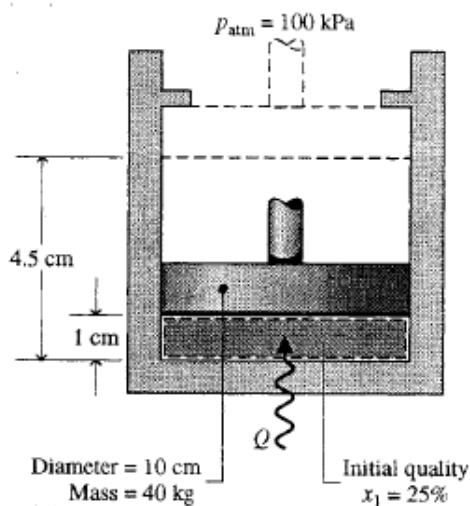
$$\Delta KE + \Delta PE + \Delta U = Q - W \Rightarrow \Delta U = 0$$

Since internal energy of an ideal gas depends on temperature alone,  $T_2 = T_1$ . Thus

$$P_2 = (5 \text{ bar}) \left[ 1 \right] \left[ \frac{0.2 \text{ m}^3}{0.5 \text{ m}^3} \right] = 2 \text{ bar}$$

### Problem 4

SCHEMATIC & GIVEN DATA:



ASSUMPTIONS: 1. The water is a closed system. 2. The pressure is constant until the piston hits the stops. 3. Friction between the piston and cylinder wall can be ignored. 4. For the system,  $\Delta PE, \Delta KE$  can be ignored. 5.  $g$  is constant.

To begin, fix the three states located by dots on the p-v diagram. State 1 is fixed by  $x_1 = 25\%$  and  $p_1$ , which is found from a force balance on the piston: The force exerted by the water on the lower face of the piston equals the piston weight plus the force exerted on the top face by the atmosphere:

$$p_1 A = m_{\text{pist}} g + p_{\text{atm}} A, \quad A = \pi D_{\text{pist}}^2 / 4$$

⇒

$$p_1 = \frac{m_{\text{pist}} g}{A} + p_{\text{atm}} = \frac{(40 \text{ kg})(9.81 \text{ m/s}^2)}{\frac{\pi}{4} (0.1 \text{ m})^2} \left| \frac{1 \text{ N}}{\text{kg} \cdot \text{m/s}^2} \right| \left| \frac{\text{bar}}{10^5 \text{ N/m}^2} \right| + (100 \text{ kPa}) \left| \frac{1 \text{ bar}}{10^2 \text{ kPa}} \right|$$

$$= 1.5 \text{ bar}$$

State a is fixed by  $p_a = p_1 = 1.5 \text{ bar}$  and the specific volume,  $v_a$ . From the given geometry,  $V_a = 4.5 V_1$  or  $v_a = 4.5 v_1$ , where

$$v_1 = v_{f1} + x_1 (v_{g1} - v_{f1}) = \frac{1.0528}{10^3} + 0.25 (1.159 - \frac{1.0528}{10^3}) = 0.29054 \text{ m}^3/\text{kg}$$

$$\Rightarrow v_a = 4.5 (0.29054) = 1.3074 \text{ m}^3/\text{kg}$$

Since  $v_a > v_g(1.5 \text{ bar})$ , state a falls in the superheated vapor region, as shown in the p-v diagram. State 3 is fixed by  $v_3 = v_a$  and  $p_3 = 3 \text{ bar}$ .

The total mass of water is

$$m = \frac{V}{v_1} = \frac{\pi/4 (0.1 \text{ m})^2 (0.01 \text{ m})}{0.29054 \text{ m}^3/\text{kg}} = 2.703 \times 10^{-4} \text{ kg}$$

Since there is no work associated with the constant volume portion of the process, the total work is obtained as the water undergoes the constant pressure expansion from 1 to a:

$$W = \int_1^a p \, dV = p [V_a - V_1] = p [V_3 - V_1] = m p (v_3 - v_1)$$

That is, with  $v_3 = 4.5 v_1$ ,

$$W = (2.703 \times 10^{-4} \text{ kg}) (1.5 \text{ bar}) \left| \frac{10^5 \text{ N/m}^2}{1 \text{ bar}} \right| (3.5) (0.29054 \frac{\text{m}^3}{\text{kg}}) \left| \frac{1 \text{ J}}{1 \text{ N} \cdot \text{m}} \right|$$

$$= 41.23 \text{ J}$$

An energy balance reads

$$\Delta U + \Delta KE + \Delta PE = Q - W$$

$$\Rightarrow Q = \Delta U + W$$

$$= m (u_3 - u_1) + W$$

where

$$u_1 = u_{f1} + x_1 (u_{g1} - u_{f1}) \\ = 466.94 + 0.25(2519.7 - 466.94) = 980.13 \text{ kJ/kg}$$

To find  $u_2$ , interpolate at 3 bar using  $v_2$ :  $u_2 = 3263.53$   
Then

$$Q = (2.703 \times 10^{-4} \text{ kg}) (3263.53 - 980.13) \frac{\text{kJ}}{\text{kg}} \left| \frac{10^3 \text{ J}}{\text{kJ}} \right| + 41.23 \text{ J} \\ = 658.43 \text{ J}$$